

NPSH and cavitation

$$NPSH_{re} < NPSH_{av}$$

$$NPSH_{re} < p + h - h_{suction} - p_h$$

$$NPSH_{re} < p_{suction} - p_h$$

$NPSH_{av}$ = difference between available inlet pressure (in suction flange) and vapour pressure of liquid being pumped

$NPSH_{re}$ = NPSH value required by the pump

p = Absolute air pressure

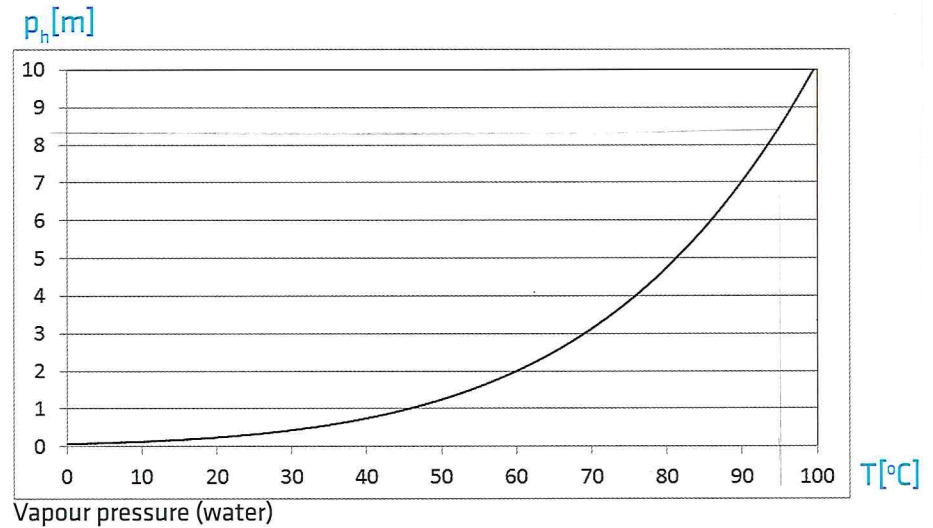
p_h = Absolute liquid vapour pressure at the operating temperature

h = Liquid geodetic suction head

$h_{suction}$ = Pressure losses in suction pipes

$p_{suction}$ = Absolute suction pressure

The $NPSH_{av}$ value of a system refers to the actual difference between inlet pressure (in the suction flange) and vapour pressure of the liquid being pumped. The $NPSH_{re}$ value required of the pump must be smaller than the $NPSH_{av}$ value in order to prevent cavitation from occurring. A safety margin of 0.5 m must be added to the measurement value.



At normal air pressure levels (10 m water column, 1,013 mbar = 760 mm Hg), clean water boils at 100°C. It can be seen from the curve that water boils at 60°C when the absolute pressure is 2 m wc (i.e. 8 m wc below atmospheric pressure). The boiling point of water at less than 40°C can be achieved at a very low pressure. Vice versa, at the top of Mount Everest, where air pressure is about 0.6 bar (6 m), water boils at +85°C.

Example:

Open tank (p = air pressure = 10 m) where the water temperature is + 90°C (p_h = 7 m), suction pipe losses 1 m and liquid suction head flange +2 m. The pump duty point 20 l/s, 7.8 m.

Is the selected pump suitable for the use in question? An example of calculation:

Pump type: AL_-1102/4/Ø188 2,2 kW

$$NPSH_{re} < p + h - h_{suction} - p_h$$

$$NPSH_{re} < 10 \text{ m} + 2 \text{ m} - 1 \text{ m} - 7 \text{ m}$$

$$NPSH_{re} < 4 \text{ m}$$

When observing the safety margin 0.5 m, the $NPSH_{re}$ value of the pump must be smaller than 3.5 m in order to prevent the pump from cavitating. $NPSH_{re}$ of pump AL_-1102/4/Ø188 = 2.7 m, whereby it will not cavitate.

